

Robotic Arm



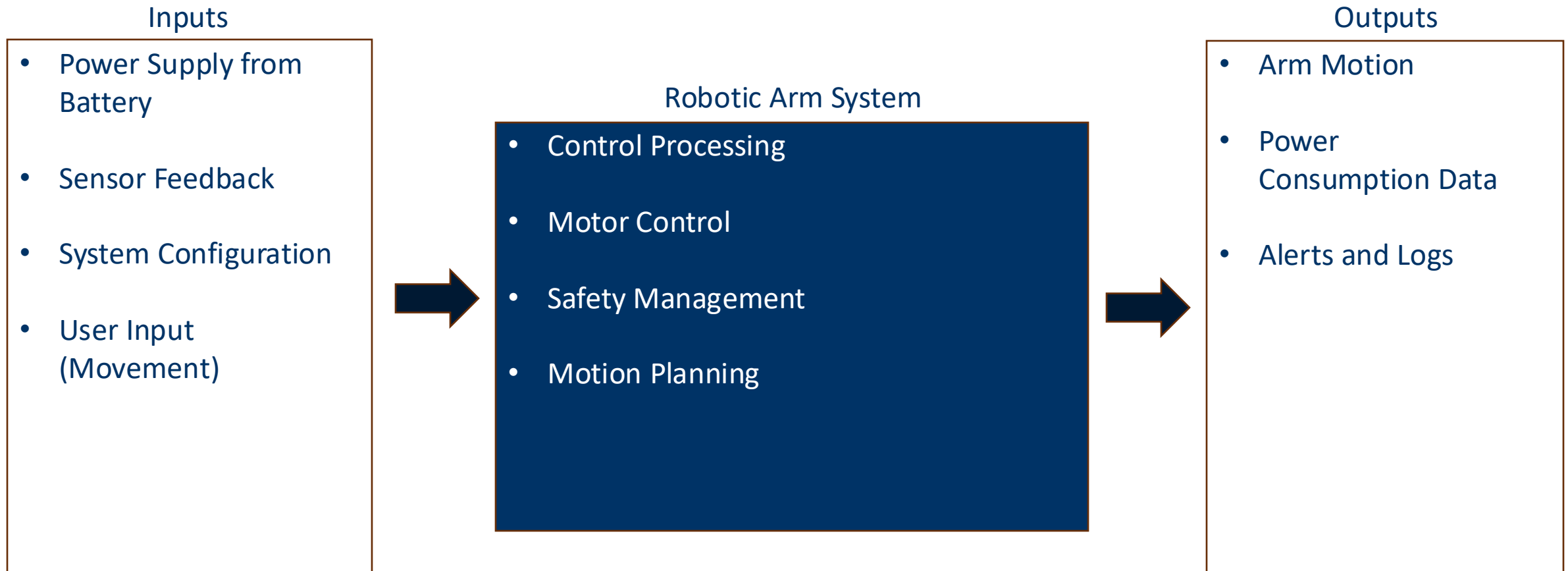
Cole Pace, Caleb Lamca, Kaitlyn Davis Joel Gisleskog, and Colin Donnellan

Project Description

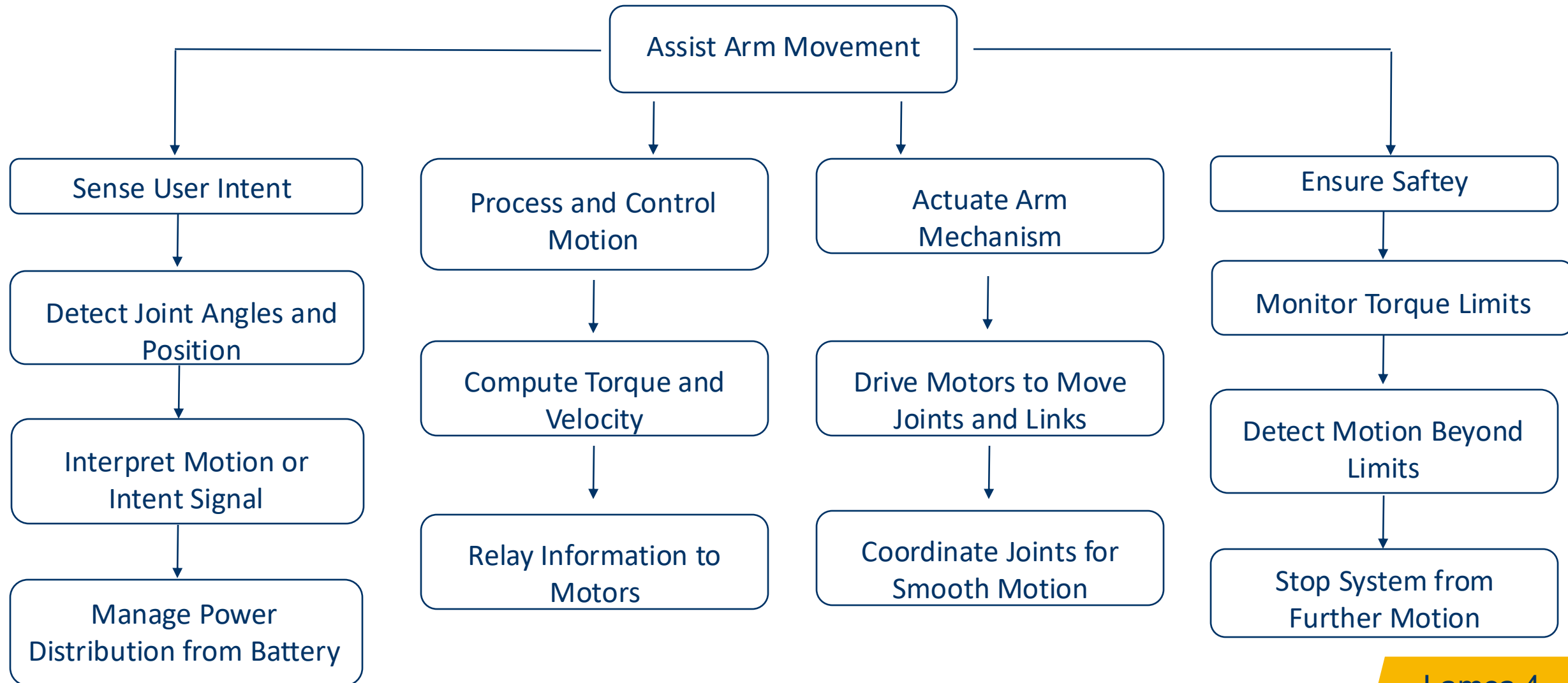
Project Description

- Stroke is the leading cause of upper limb disability.
- Survivors often lose mobility in one arm, limiting daily activities.
- Goal: develop a **waist-mounted robotic arm** that
 - Offers **active gravity compensation**
 - Remains **lightweight, low-profile, and energy efficient**
 - Enables the arm to rest naturally by the user's side.
- **Client:** Dr. Zach Lerner, Associate Professor of Mechanical Engineering, NAU.
- **Sponsorship:** W.L Gore

Black Box Model



Functional Decomposition



Concept Generation

- We generated/evaluated 3 different components of our device.
 - Motor
 - 3 different motors were given to us by our client to evaluate.
 - Joints
 - Standard mechanical joints were researched and DOF were considered with end goal of the device.
 - Link Geometry
 - Different simple cross-sections were considered and later evaluated.

Concept Generation - Motor

Model	Rated Voltage (V)	Rated Power (W)	Rated Torque (Nm)	Rated Current (A)	Rated Speed (RPM)	Peak Torque (Nm)	Peak Current (A)	No-load Speed (RPM)	Reduction Ratio	Weight (G)	Size (diameter *length) MM	Driver Board	Encoder
AK45-36 KV80	24	33	8	2	40	24	6.5	52	36:1	340	φ55*54	Yes	Single
AK45-10 KV75	24	39	2.5	2.1	150	7	5	180	10:1	260	φ53*43	Yes	Single
AK40-10 KV170	24	59	1.3	2.7	370	4.1	7.3	435	10:1	200	φ53*37	Yes	Single



Figure 1: AK40-10 KV170 [1]



Figure 2: A45-10 KV75 [2]



Figure 3: AK45-36 KV80 [3]

Concept Generation - Joints

- Different mechanical joint designs with varying DOF and geometries were researched.

- **Ring Joint (2 DOF)**

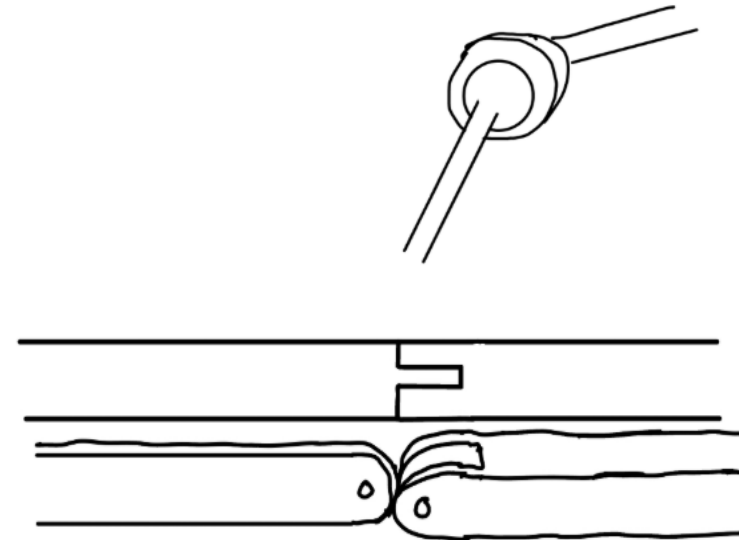
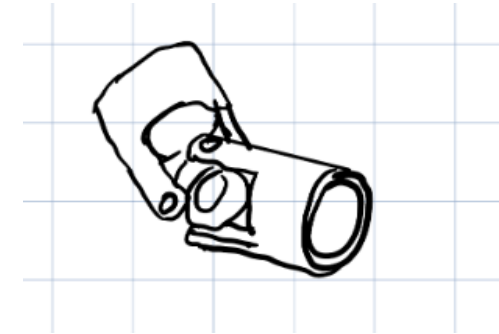
- **Pros:** Allows for finer movements of the hand/arm.
- **Cons:** Transmitting power to two different axis would increase cost and complexity.

- **Ball Joint (3 DOF)**

- **Pros:** Allows for smoother movement of the arm.
- **Cons:** Requires three motors for each DOF.

- **Revolute Joint (1 DOF)**

- **Pros:** Simplifies transmission, only requires one motor.
- **Cons:** Limited DOF leads to less smooth motion.



Concept Generation – Link Geometry

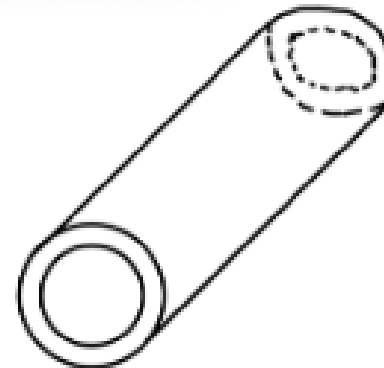
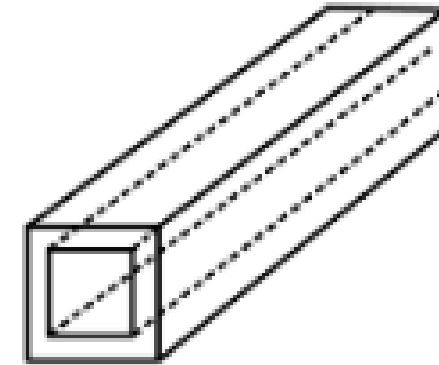
- Two different cross-sectional areas were evaluated.

- **Hollow Rectangular:**

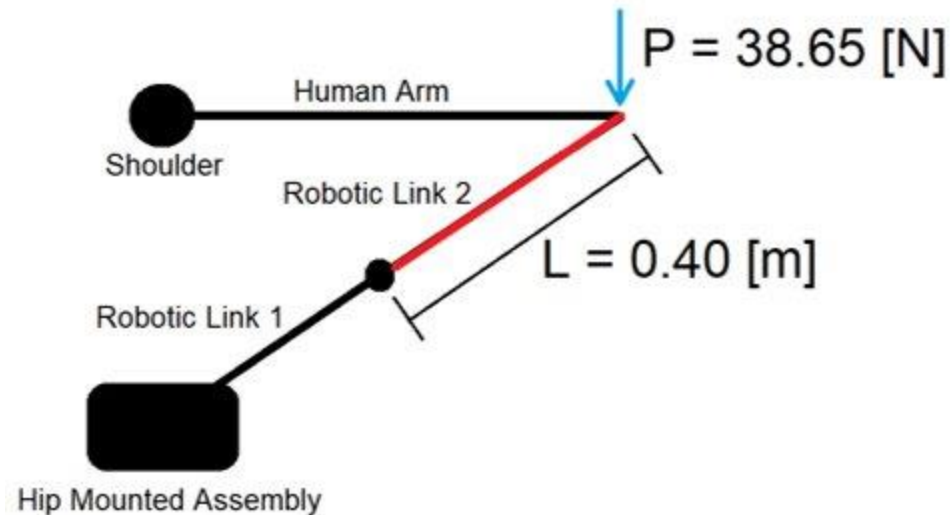
- **Pros:** Strong directional stiffness while keeping weight low.
- **Cons:** Weak in torsion and off-axis bending.

- **Hollow Circular:**

- **Pros:** Resists twisting far better than rectangular.
- **Cons:** Less stiffness per unit weight in one direction compared to rectangular.



Bending Stress of Varying Beam Geometries in Link 2



- What will be the bending stress in linkage 2 given different geometries?

Compared 2 geometries; a circular tube (c), and a square tube (s).

Material was chosen as 6061-T6 Aluminum due to its lightweight and higher strength properties.

Both geometries are strong enough for the application, however, the circular tube is over 4times stronger.

Achieving a FoS > 3 is only achievable with the circular cross-sectional geometry.

See Appendix A for assumptions, technical variables, and supporting calculations

- $$\sigma_{\max_c} = \frac{M_{\max} \cdot c}{I_c} = 15.4 \text{ [MPa]} \leq 80.33 \text{ [MPa]}$$

- $$\sigma_{\max_s} = \frac{M_{\max} \cdot c}{I_s} = 69.4 \text{ [MPa]} \leq 80.33 \text{ [MPa]}$$

- $$\sigma_{\text{allowable}} = \frac{\sigma_{\text{yeild}}}{FoS} = 80.33 \text{ [MPa]}$$

Capacity Required

What is the capacity needed for each of the three motors to last for 8 hours?

$$\text{Capacity} = \frac{\text{Rated Power}}{\text{Rated Voltage}} \cdot 8 \text{ hours}$$

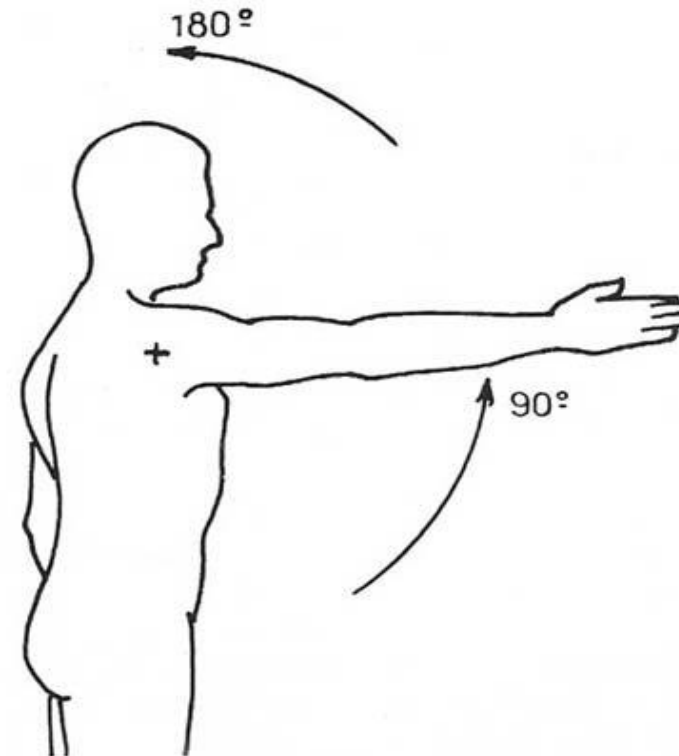
AK45-36: 11Ah

AK45-10: 13Ah

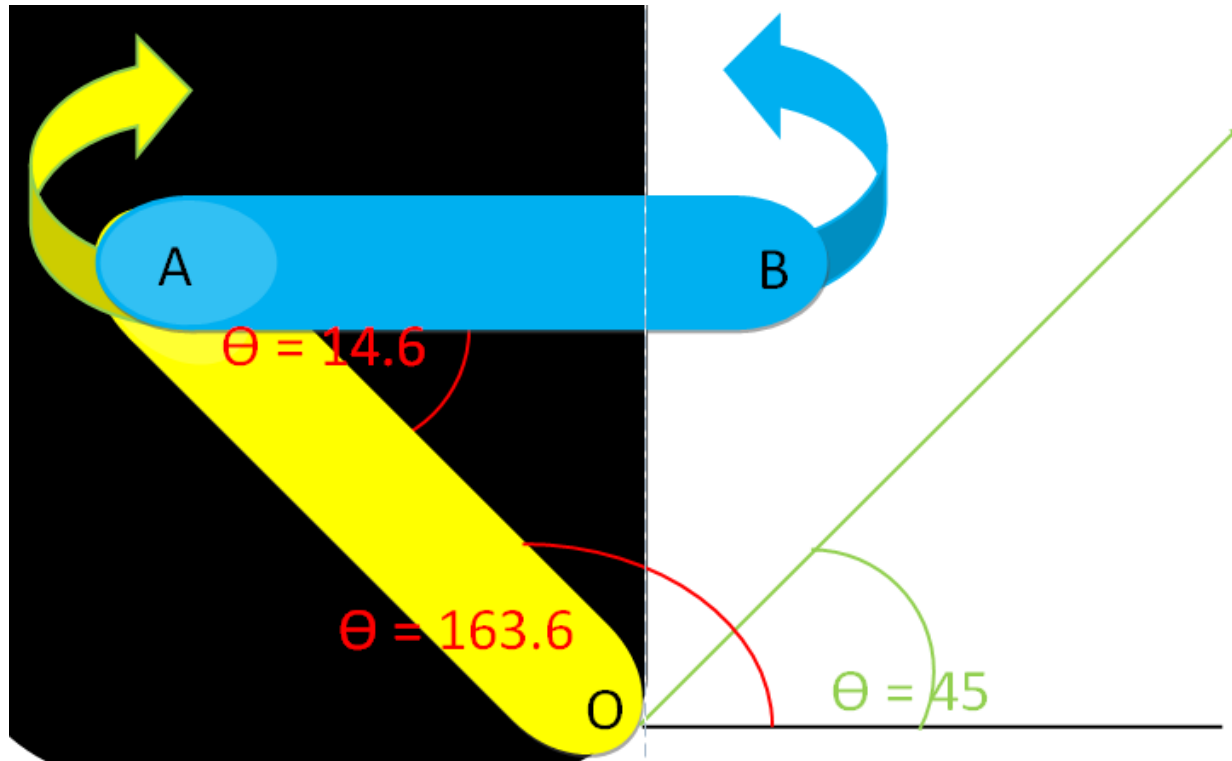
AK40-10: 20Ah

Arm Motion

- Arm (shoulder flexion) velocity
- Shoulder flexion from hanging straight down (0 degrees) to straight forward (90 degrees)
- Average shoulder to elbow length: 330mm (13in)
- Average angular velocity: $w_{avg} = \frac{\Delta \theta}{Time\ t}$
- Using $\Delta \theta = 90$ degrees and time 1.28 seconds.
- $w_{avg} = \frac{\Delta \theta}{Time\ t} = 1.227 \frac{rad}{s}$
- Linear velocity (elbow): $v = w_{avg} * r = 0.405 \frac{m}{s}$



Link Motion



$$O = (0,0)$$

$$A_s = (-238\text{mm}, 70\text{mm})$$

$$B_s = (10\text{mm}, 70\text{mm})$$

$$\text{link 1} = 248\text{mm}$$

$$\text{link 2} = 248\text{mm}$$

$$\text{distance from elbow to waist} = 70\text{mm}$$

$$\sin(\theta) = \left(\frac{\text{opp}}{\text{hyp}}\right)$$

$$\theta = 14.6 \text{ degrees}$$

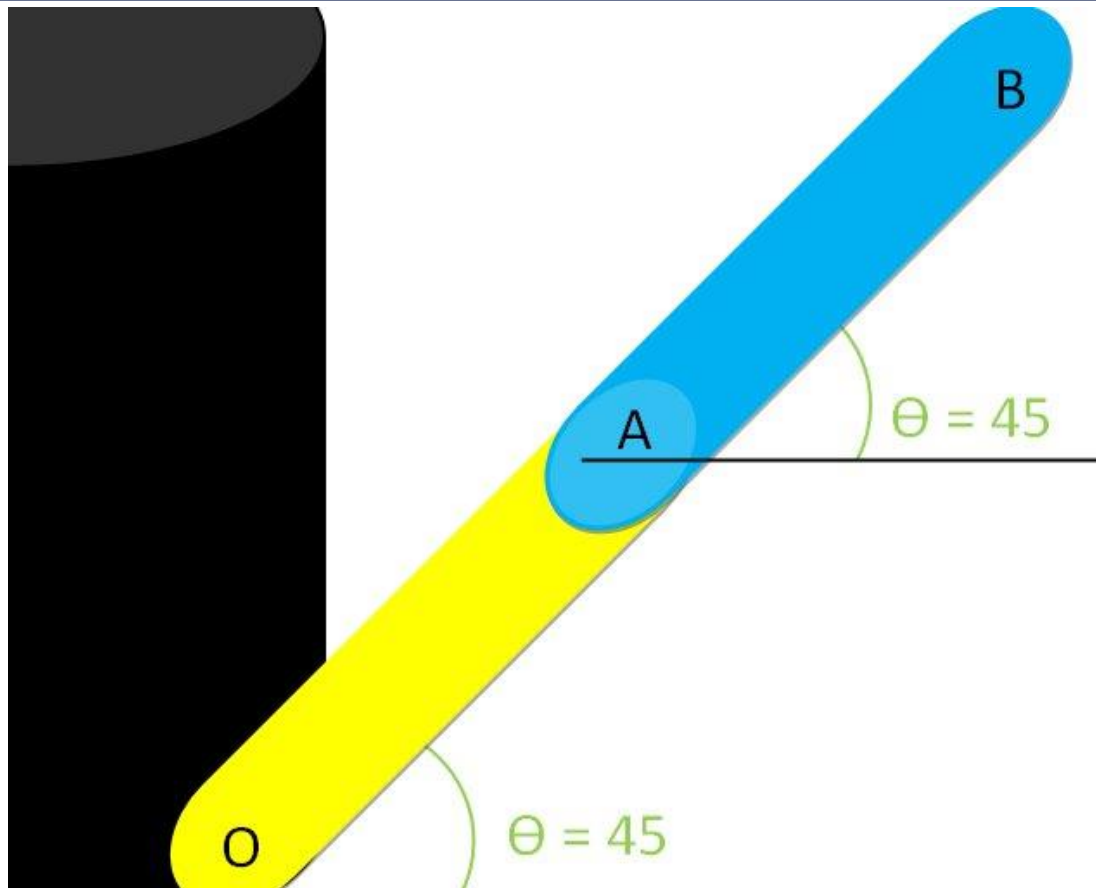
$$l_1 x_1 = l * \cos(\theta) = -237.9\text{mm}$$

$$l_1 y_1 = l * \sin(\theta) = -70\text{mm}$$

$$B_{x1} = 10\text{mm}$$

$$B_{y1} = 10\text{mm}$$

Link Motion



$$O = (0,0)$$

$$A_f = (175.36\text{mm}, 175.36\text{mm})$$

$$B_f = (350.7\text{mm}, 350.7\text{mm})$$

new position at 45 degrees

$$l_1x_2 = l * \cos(\theta) = 175.36\text{mm}$$

$$l_1y_2 = l * \cos(\theta) = 175.36\text{mm}$$

$$B_{x2} = 350.7\text{mm}$$

$$B_{y2} = 350.7\text{mm}$$

time wanted to reach through 90 degrees = 1.28s

$$\theta_1 = 163.6 - 45 = 118.6 \text{ degrees}$$

$$\theta_2 = 0 + 45 = 45 \text{ degrees}$$

$$\omega_1 = \frac{\Delta\theta_1}{1.28} = -1.6171 \text{ rad/s}$$

$$\omega_2 = \frac{\Delta\theta_2}{1.28} = +0.6136 \text{ rad/s}$$

Torque at Joints

- Remote Transmission

- Hip:

$$\tau_1 = g[m_1 r_1 + m_2 (L_1 + r_2) + m_p (L_1 + L_2)] + \alpha_1 [m_1 r_1^2 + m_2 (L_2 + r_2)^2 + m_p (L_1 + L_2)^2]$$

$$\tau_1 = 6.78 \text{ N} \cdot \text{m}$$

$$\tau_{1S} = 10 \text{ N} \cdot \text{m}$$

- Elbow:

$$\tau_2 = g[m_2 r_2 + m_p L_2] + \alpha_2 [m_2 r_2^2 + m_p L_2^2]$$

$$\tau_2 = 2.94 \text{ N} \cdot \text{m}$$

$$\tau_{2S} = 4.5 \text{ N} \cdot \text{m}$$

Torque at Joints

- Direct Drive

- Hip:

$$\tau_1 = g[m_1 r_1 + (m_2 + m_{m2})(L_1 + r_2) + m_p(L_1 + L_2)] + \alpha_1 \left[m_1 r_1^2 + m_2 (L_2 + r_2)^2 + m_p (L_1 + L_2)^2 \right]$$

$$\tau_1 = 7.65 \text{ N} \cdot \text{m}$$

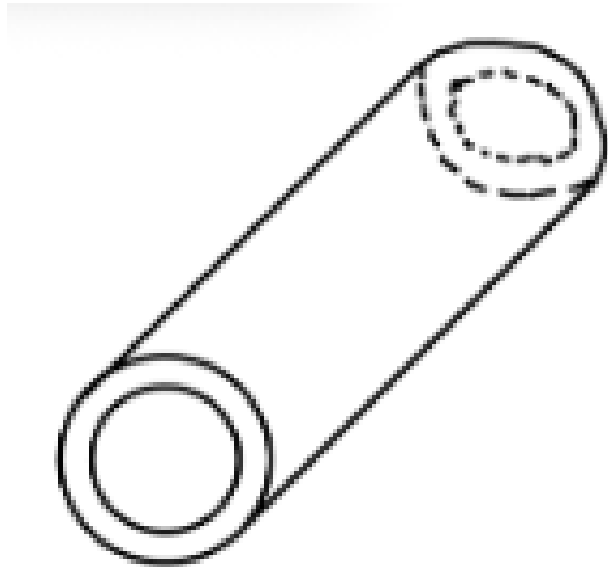
$$\tau_{1S} = 11.5 \text{ N} \cdot \text{m}$$

Concept Evaluation

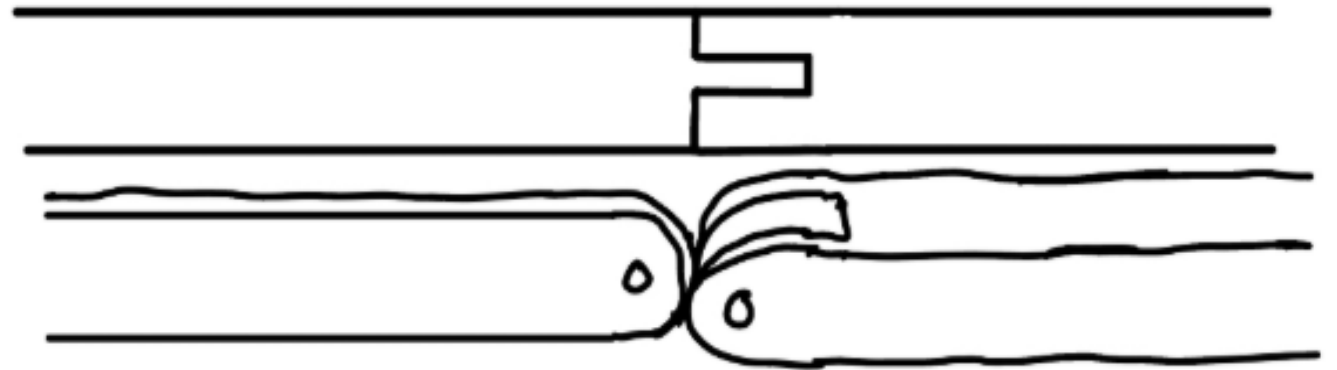
- We evaluated our sub-topics against our customer and engineering requirements.
 - Range of Motion: Determined by the joint selection. (3DOF)
 - Ease of Use: Determined by motor and joint selections.
 - Will be quantified by time to complete tasks.
 - Durability: Determined by link structure.
 - Will be quantified by stress analysis once more aspects of the device are determined.
 - Low-Profile: Combinations of joint structure, motor, and link structure.
 - Will be quantified by weight and the space that the device occupies.

Joint and Link Selection

The hollow circular tube were chosen for being stronger than the rectangular tube and would be able to hit a factor of safety of 3.



The revolute joint was chosen by being able to have the movement wanted as well as being easiest to handle the Bowden cables needed for a transmission system.



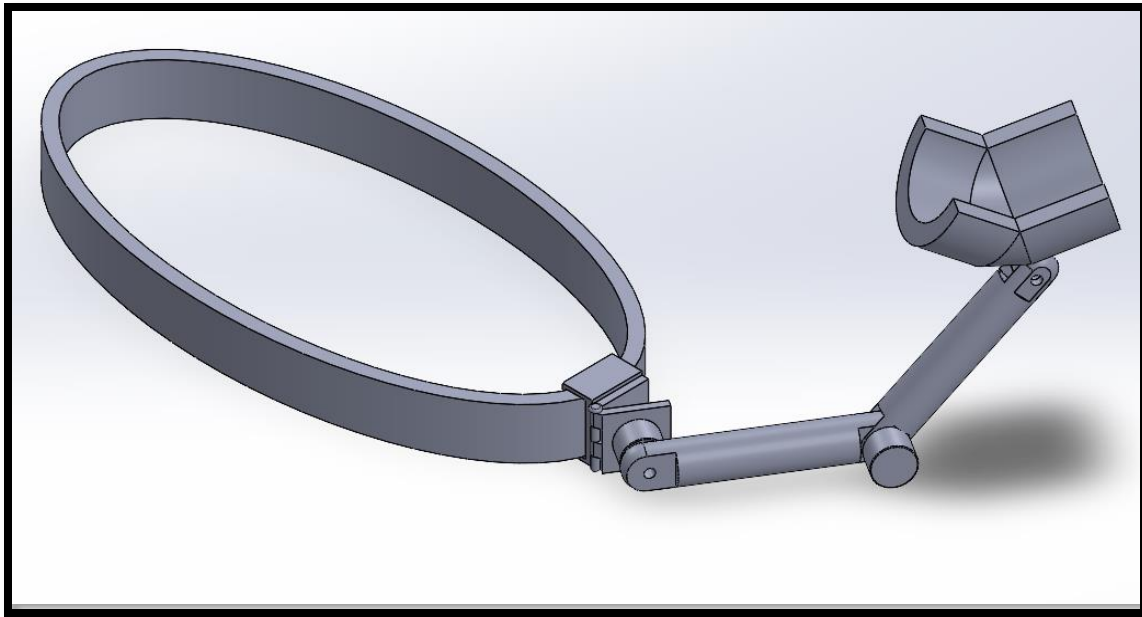
Motor Selection

- We ultimately decided on:
 - AK45-36 KV80 Motor from CubeMars
 - Rated torque: 8 N*m
 - Peak Torque: 24 N*m
 - Can handle the torques for each joint in both a direct drive and transmission design.

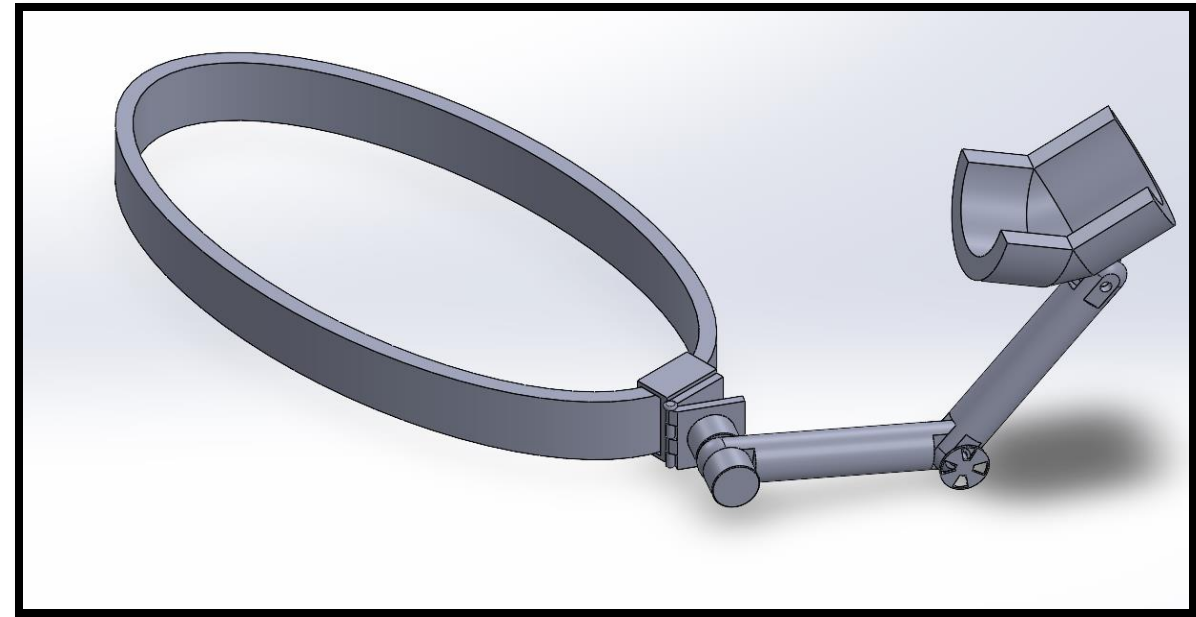


Model

Direct Drive



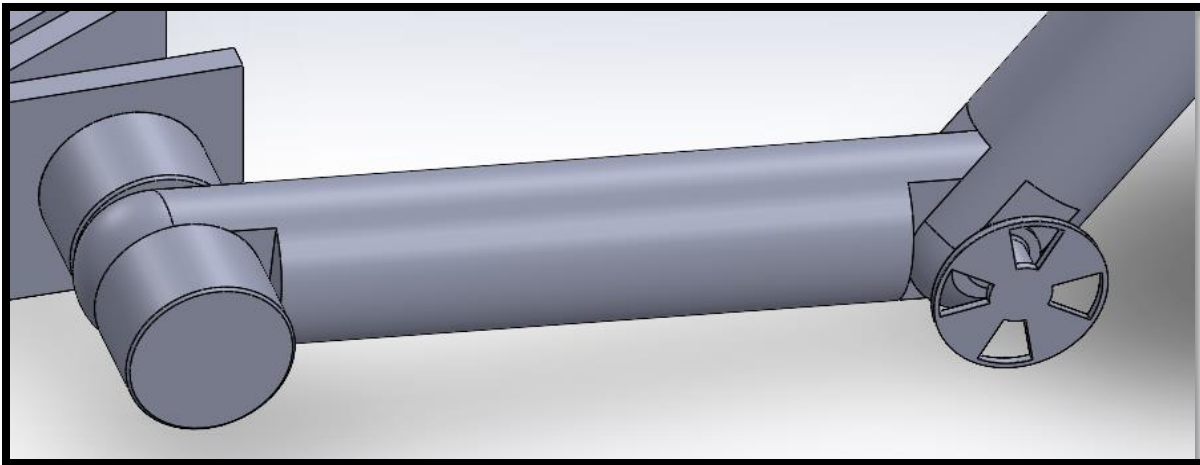
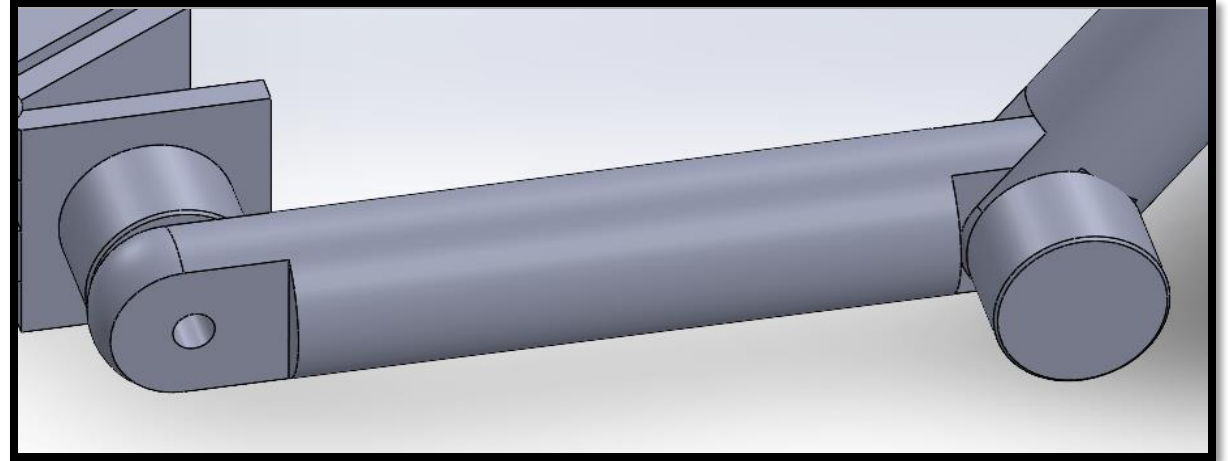
Transmission



Model

Direct Drive System:

- 2 Motors, 1 at each link
- Motor directly transmits power
- Weight further from body - increases the required output torque of the motor
- Less complex system – fewer moving parts



Transmission System:

- 2 Motors at the hip
- Pulley with cable to transmit power
- Weight kept close to body – reduces the required output torque of the motor
- More complex system – more moving parts

Bill of Materials

Item	Price
2 AK45-36 motor	\$371.80
3D filament	\$100
Battery	\$300
Waist belt	\$50
Hinge	\$15

Budget

- Funding from W.L Gore: \$4000
 - NAU 5% processing fee: -\$200
 - Fundraising (at least %10): \$400
 - Total Est. Budget: \$4200
-
- We have an estimate of \$3975 for total cost of possible items. The team will have a remaining balance of \$225. We will need to fundraise more to begin prototyping.

Expenses		
Category	Items(s)	Cost
Tools and materials:	3D printer Parts	\$100
	3D printer Filament	\$100
Manufacturing:		\$300
Parts:	Motors	\$375
	Battery	\$300
	Miscellaneous Parts	\$700
Prototyping:	1st	\$1200
	2nd	\$900
TOTAL:		\$3975

Fundraising

- Need to accumulate 10% of the \$4,000 budget for a minimum of \$400 total
- In talks with multiple companies regarding sponsorships, services, or cash donations
- We plan to fundraise the entire 10% on or before week 8, or the testing of prototype 1



Schedule

Plan duration

Actual Start

Completed

Beyond Completion

	August				September				October				November				December			
				wk1	wk2	wk3	wk4	wk5	wk6	wk7	wk8	wk9	wk10	wk11	wk12	wk13	wk14	wk15	wk16	
Requirements/ research																				
Equations																				
Presentation 1																				
Conceptual designs																				
Fundraising																				
Presentation 2																				
Begin modelling																				
Testing prototype 1																				
Presentation 3																				
1st Prototype Demo																				
Begin prototype 2																				
Testing for Prototype 2																				
2nd Prototype Demo																				

**Thank you
And
Any Questions?**

Appendix

- [1] *Cubemars.com*, 2025. <https://www.cubemars.com/product/ak40-10-robotic-actuator.html>
- [2] “AK45-10 Robotic Actuator – 10:1 Gear Ratio, 260g, $\Phi 53 \times 43$ mm,” *CubeMars*, 2025. <https://www.cubemars.com/product/AK45-10-robotic-actuatuor.html>
- [3] “AK45-36 Robotic Actuator – Ultra-High Torque, 36:1 Gear Ratio,” *CubeMars*, 2025. <https://www.cubemars.com/product/AK45-36.html>

Appendix A

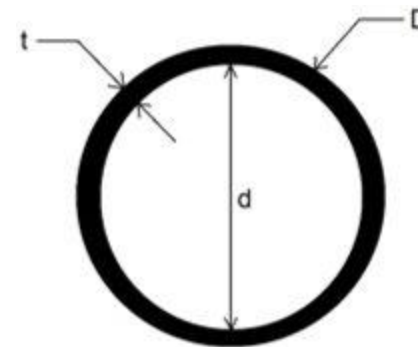
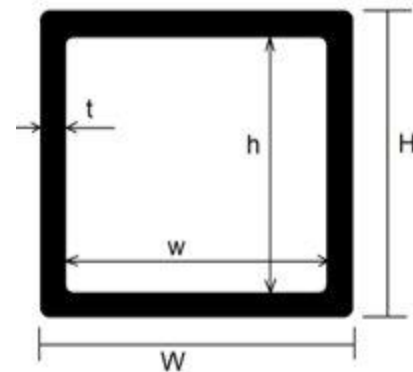
Broad Assumptions:

- Average Human male arm mass = 1.89 [kg]
 - Average total body mass of 75 [kg]
 - Forearm Mass + Wrist Mass = 2.52% of Total Body Mass
- Arm is fully outstretched in front of the body
- Linkage at 45 degrees
- Treat connection to link 1 as fixed
- Robotic linkage is split into 2 parts of equal length (0.40 [m])
- Deflection due to Axial compression is negligible

Technical Variables:

- | | |
|---|-----------------------|
| • Material: 6061-T6 Aluminum | • $c = 0.015$ |
| • $FoS = 3$ | • $t = .003 [m]$ |
| • $g = 9.81 \left[\frac{m}{s^2} \right]$ | • $\theta = 45^\circ$ |
| • $m_{total} = 75 [kg]$ | • $D = 0.254 [m]$ |
| • $\sigma_{yeild} = 241 [MPa]$ | • $d = 0.0251 [m]$ |
| • $a = 0.0525$ | • $H = 0.0254 [m]$ |
| • $E = 6.89 \cdot 10^{10} [Pa]$ Elastic Modulus | • $W = 0.0254 [m]$ |
| • $L = 0.40 [m]$ | • $h = 0.0251 [m]$ |
| | • $w = 0.0251 [m]$ |

Geometries:



Calculations:

- $c = \frac{D}{2} = \frac{H}{2} = 0.0127 [m]$
- $m_{arm} = m_{total} \cdot a = 3.94 [kg]$
- $P = m_{arm} \cdot g = 38.65 [N]$
- $P_{\perp} = P \cdot \cos \theta = 27.33 [N]$
- $M_{max} = P_{\perp} L = 10.93 [Nm]$
- $I_c = \frac{\pi}{64} (D^4 - d^4) = 9 \cdot 10^{-6} [m^4]$
- $I_s = \frac{HW^3 - hw^3}{12} = 2 \cdot 10^6 [m^4]$
- $\delta_{\perp c} = \frac{P_{\perp} L^3}{3EI_c} = .00094 [mm]$
- $\delta_{\perp s} = \frac{P_{\perp} L^3}{3EI_s} = .004 [mm]$